

Fig. 4 Influence of tunnel noise on pressure fluctuation spectra.

layer at the same freestream Mach number and with  $\bar{p}/q_\infty \approx 0.015$ . The turbulence intensity levels in the separated layer indicate an amplification factor of 3-4 which is of the same order as that for the pressure fluctuation levels. It is observed that there are no significant changes in the turbulence levels with the change in the tunnel noise level from  $\bar{p}/q_\infty = 0.010$  to 0.015. The freestream level of turbulence is unaffected by separation and change in the tunnel noise.

The spectra of pressure fluctuation in the region of shock/boundary-layer interaction for a tunnel noise level  $\bar{p}/q_\infty = 0.01$  are shown in Fig. 3. The increase in pressure fluctuation levels in the separated region essentially occur at higher frequencies. The peak frequencies shift progressively across the shock. The peak frequency based upon the separation bubble length is  $n_b = 2$ . The peak frequencies based upon the chord length occur at  $n \approx 4$  with harmonics at  $n \approx 8$  and  $n \approx 12$ . The peak frequencies upstream and downstream of the interaction ( $x/c = -0.75$  and 2.25) are also  $n \approx 4$ . This suggests a correlation between the tunnel noise and the pressure fluctuation levels in the separated layer.

As shown in Fig. 4, changes in the wind-tunnel test section configuration to produce different noise levels have an effect on the pressure fluctuation spectra within the region of shock/boundary-layer interaction but do not effect the length of the separation bubble.

### Acknowledgment

This work originated as part of the research supported by the Science Research Council contract GR/A/53772. The help from the Aeronautical Engineering workshop is greatly appreciated.

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AIAA 81-4040

## Algorithm for Rapid Integration of Turbulence Model Equations on Parabolic Regions

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### Introduction

THE advent of advanced engineering models of turbulence such as those described by Launder and Spalding<sup>1</sup> has been accompanied by a need for advanced numerical methods. For most of these advanced models, conventional numerical methods often are quite difficult to apply and sometimes do not work at all. For example, for a relatively simple flow such as fully developed channel flow explicit time-marching algorithms, relaxation schemes and the shooting method fail to yield a converged solution. The purpose of this Note is to cite yet another occasion where integration of an advanced set of turbulence-model equations has been unusually difficult and to describe the resolution of the problems encountered.

While developing a three-dimensional boundary-layer program using a standard parabolic marching scheme, the author has found computing time with the Wilcox-Rubesin<sup>2</sup> two-equation turbulence model to be very lengthy. The long computing time occurs because converged solutions are possible only when very small streamwise steps are taken. This observation is consistent with results obtained by Rastogi and Rodi<sup>3</sup> who have devised a three-dimensional boundary-layer program which uses the Jones-Launder<sup>4</sup> two-equation turbulence model. Rastogi and Rodi find that their initial stepsize  $\Delta s$  must be of the order of 1/100th of the local boundary-layer thickness  $\delta$  and that ultimately  $\Delta s$  generally cannot exceed  $\delta/2$ . By contrast, a typical mixing-length computation for a three-dimensional wing begins with  $\Delta s/\delta \sim 4$  and maintains a  $\Delta s/\delta$  ratio in excess of unity. Such a severe stepsize limitation would increase computing time over that required for the mixing-length model by a factor of 10-100, depending upon the ultimate Reynolds number. Even if such run times were tolerable, the computer storage requirements for a real airplane wing would be prohibitive. Clearly an innovation is needed. The next section presents the necessary innovation.

### Analysis

#### Model Equations

To illustrate the essence of the problems encountered while integrating turbulence-model equations, we begin by casting the turbulent energy equation in discretized form for a two-dimensional boundary layer. For the Wilcox-Rubesin model the differential equation for turbulent energy  $e$  is

$$u \frac{\partial e}{\partial s} + v \frac{\partial e}{\partial y} = \gamma^* \frac{(\partial u / \partial y)^2}{\omega} e - \beta^* \omega e + \frac{\partial}{\partial y} \left\{ (\nu + \sigma^* \epsilon) \frac{\partial e}{\partial y} \right\} \quad (1)$$

where  $s$  is arclength along the body,  $y$  the distance normal to the surface,  $u$  and  $v$  the velocity components in the  $s$  and  $y$  directions, and  $\omega$  the turbulent dissipation rate. The quantities  $\nu$  and  $\epsilon$  denote kinematic molecular and eddy viscosity, respectively, while  $\gamma^*$ ,  $\beta^*$ , and  $\sigma^*$  are closure coefficients whose values have been empirically found to be 1, 9/100, and

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$\frac{1}{2}$ , respectively. The dissipation rate satisfies a similar equation; for present purposes it is necessary to focus only upon Eq. (1), and we thus omit the equation for  $\omega$ .

#### Discretization

Proceeding now to discretization of Eq. (1) we assume, for simplicity, that the mesh points are equally spaced in both the  $s$  and  $y$  directions. Conventional, second-order accurate, discretization approximations for boundary-layer parabolic marching schemes are as follows:

$$u \frac{\partial e}{\partial s} \doteq \frac{u}{\Delta s} (3e_{m+1,n} - 4e_{m,n} + e_{m-1,n}) \quad (2)$$

$$\left( v - \sigma^* \frac{\partial \epsilon}{\partial y} \right) \frac{\partial e}{\partial y} \doteq \frac{v - \sigma^* \partial \epsilon / \partial y}{2\Delta y} (e_{m+1,n+1} - e_{m+1,n-1}) \quad (3)$$

$$\left\{ \gamma^* \frac{(\partial u / \partial y)^2}{\omega} - \beta^* \omega \right\} e \doteq \left\{ \gamma^* \frac{(\partial u / \partial y)^2}{\omega} - \beta^* \omega \right\} e_{m+1,n} \quad (4)$$

$$(v + \sigma^* \epsilon) \frac{\partial^2 e}{\partial y^2} \doteq \frac{v + \sigma^* \epsilon}{(\Delta y)^2} (e_{m+1,n+1} - 2e_{m+1,n} + e_{m+1,n-1}) \quad (5)$$

where  $e_{m,n}$  denotes the value of  $e$  at  $s=s_m$  and  $y=y_n$ . Quantities without subscripts are assumed known during the (typically) iterative solution procedure. Substituting Eqs. (2-5) into Eq. (1) and regrouping terms leads to the conventional tridiagonal matrix system as follows:

$$A_n e_{m+1,n-1} + B_n e_{m+1,n} + C_n e_{m+1,n+1} = D_n \quad (6)$$

where  $A_n$ ,  $B_n$ ,  $C_n$ , and  $D_n$  are defined by

$$A_n = - \left\{ \frac{v - \sigma^* \partial \epsilon / \partial y}{\Delta y} + \frac{v + \sigma^* \epsilon}{(\Delta y)^2} \right\} \quad (7)$$

$$B_n = 3 \frac{u}{\Delta s} + 2 \frac{v + \sigma^* \epsilon}{(\Delta y)^2} - \gamma^* \frac{(\partial u / \partial y)^2}{\omega} + \beta^* \omega \quad (8)$$

$$C_n = \frac{v - \sigma^* \partial \epsilon / \partial y}{\Delta y} - \frac{v + \sigma^* \epsilon}{(\Delta y)^2} \quad (9)$$

$$D_n = \frac{u}{\Delta s} \{ 4e_{m,n} - e_{m-1,n} \} \quad (10)$$

#### Lack of Diagonal Dominance

Now, in order to have a diagonally dominant system, the condition

$$B_n \geq -(A_n + C_n) \quad (11)$$

must be satisfied. Substituting Eqs. (7-9) into Eq. (11) yields the following condition:

$$3 \frac{u}{\Delta s} - \gamma^* \frac{(\partial u / \partial y)^2}{\omega} + \beta^* \omega \geq 0 \quad (12)$$

When dissipation exceeds production, so that  $\beta^* \omega > \gamma^* (\partial u / \partial y)^2 / \omega$ , Eq. (12) is satisfied so long as we march in the direction of flow (i.e., so long as  $u$  and  $\Delta s$  are of the same sign). The system is then said to be unconditionally stable. However, when production exceeds dissipation, we have the following limit on stepsize:

$$\Delta s \leq \frac{3u\omega}{\gamma^* (\partial u / \partial y)^2 - \beta^* \omega^2} \quad (13)$$

To test the validity of Eq. (13), results of an incompressible flat-plate boundary-layer (FPBL) computation have been analyzed. At a plate-length Reynolds number  $Re_s$  of  $1.2 \times 10^6$  we have found empirically that stable computation can be achieved provided we do not exceed a Reynolds number based on  $\Delta s$  of  $2.2 \times 10^4$ , which corresponds to a ratio of  $\Delta s$  to boundary-layer thickness  $\delta$  of 1.15. Figure 1 shows  $Re_{\Delta s}$  as predicted by Eq. (13) throughout the boundary layer. As shown, the minimum value of  $Re_{\Delta s}$  according to Eq. (13) is  $1.9 \times 10^4$  and occurs at  $y/\delta \approx 0.012$ . This close agreement shows that the source of instability is lack of diagonal dominance in the tridiagonal matrix system defined in Eqs. (6-10).

#### A New Algorithm

To remedy this situation, we note first that because of nonlinearity, Eq. (6) always requires an iterative solution. Letting superscript  $i$  denote iteration number, we replace  $B_n$  and  $D_n$  by the following discretization approximations:

$$B_n = 3 \frac{u}{\Delta s} + 2 \frac{v + \sigma^* \epsilon}{(\Delta y)^2} - \gamma^* \frac{(\partial u / \partial y)^2}{\omega} + (1 + \psi_e) \beta^* \omega \quad (8a)$$

$$D_n = \frac{u}{\Delta s} \{ 4e_{m,n} - e_{m-1,n} \} + \psi_e \beta^* \omega e_{m+1,n}^{i-1} \quad (10a)$$

where  $\psi_e$  will be defined below. Then Eq. (6) is replaced by:

$$A_n e_{m+1,n-1}^{i-1} + B_n e_{m+1,n}^{i-1} + C_n e_{m+1,n+1}^{i-1} = D_n \quad (6a)$$

Inspection of Eqs. (6a), (8a) and (10a) shows that when convergence has been achieved (i.e., when  $e_{m,n}^i$  and  $e_{m,n}^{i-1}$  differ by a negligible amount), the terms on the right- and left-hand sides which are proportional to  $\psi_e$  cancel identically. Hence, the basic equation has in no way been modified. The advantage of this procedure becomes obvious upon inspection of the stability condition which now becomes:

$$\frac{3u}{\Delta s} \geq \frac{\gamma^* (\partial u / \partial y)^2}{\omega} - (1 + \psi_e) \beta^* \omega \quad (12a)$$

Clearly,  $\psi_e$  can be chosen to insure that the right-hand side of Eq. (12a) is always negative which corresponds to *unconditional stability*, i.e., no limit on streamwise stepsize. Note that a quantity analogous to  $\psi_e$  must be defined (in a similar way) to insure stable integration of the turbulent dissipation rate equation.

#### Specification of $\psi_e$

Numerical experimentation has shown that the best results are obtained when  $(1 + \psi_e) \beta^* \omega$  exceeds  $\gamma^* (\partial u / \partial y)^2 / \omega$  by about 30%, a condition which is insured by defining  $\psi_e$  as follows:

$$\psi_e = \begin{cases} 3/10 & , \gamma^* (\partial u / \partial y)^2 \leq \beta^* \omega^2 \\ \gamma^* (\partial u / \partial y)^2 / \beta^* \omega^2 - 7/10 & , \gamma^* (\partial u / \partial y)^2 > \beta^* \omega^2 \end{cases} \quad (14)$$

It is interesting to note that numerical experimentation also shows that if  $\psi_e$  is too large, say  $\psi_e = 2$ , stable integration is inhibited. This phenomenon results from the nonlinear interaction between the streamwise convection term and the source terms in the  $\omega$  equation. This can be shown by performing a linear stability analysis on the following limiting form of the Wilcox-Rubesin turbulent dissipation rate equation:

$$u \frac{\partial \omega^2}{\partial s} = \gamma \left( \frac{\partial u}{\partial y} \right)^2 \omega - \beta \omega^3 \quad (15)$$

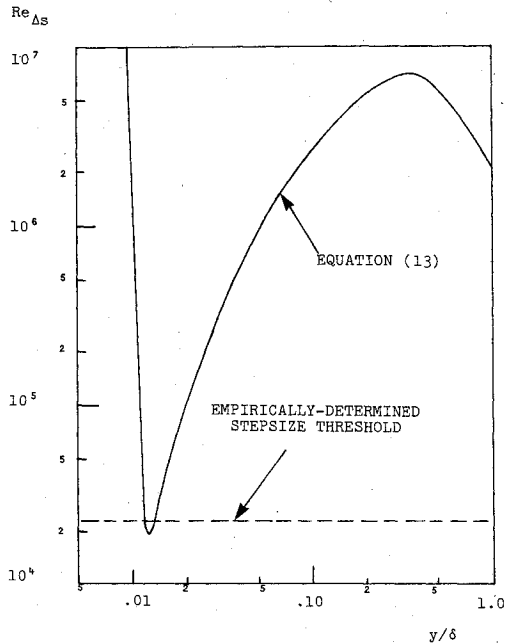


Fig. 1 Comparison of theoretical and empirically determined stepsize threshold for a flat-plate boundary layer,  $Re_s = 1.2 \times 10^6$ .

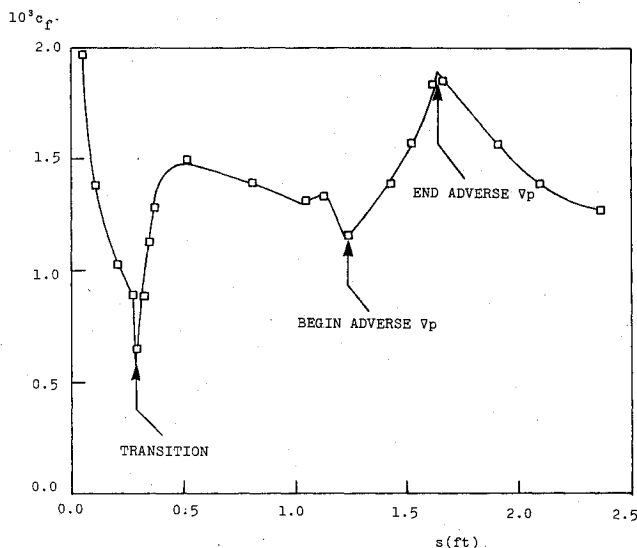


Fig. 2 Comparison of computed skin friction for Lewis-Gran-Kubota Mach 3 boundary layer with adverse and favorable pressure gradients.

where  $\gamma = 1/3$  and  $\beta = 3/20$ . It is easy to show that solution errors will diminish from one iteration to the next provided the  $\omega$  equation analog to  $\psi_e$ , viz,  $\psi_\omega$  satisfies the following condition:

$$\psi_\omega < 1 + \frac{\gamma(\partial u / \partial y)^2}{\beta \omega^2} \quad (16)$$

which is consistent with results of the numerical experimentation noted above. In all of the applications presented below, the assumption has been made that the quantities  $\psi_e$  and  $\psi_\omega$  are equal so that we write

$$\psi_\omega = \psi_e \quad (17)$$

### Applications

Using the new algorithm, the FPBL computation has been repeated with larger  $\Delta s$ . With negligible sacrifice in accuracy, stable computation is possible up to an increase of a factor of 8. Stable computation is possible when  $\Delta s$  increases by a factor of 16, although numerical oscillations begin to develop because of insufficient resolution.

To test the new algorithm further, the Mach 3 compressible boundary-layer experimentally documented by Lewis et al.<sup>5</sup> has been computed first by using 680 steps in the streamwise direction and then by using 68 steps (so that  $\Delta s$  increases by a factor of 10). Figure 2 compares computed skin friction  $c_f$  for the two computations. As shown, computed  $c_f$  values are generally within 1-2% with a peak difference of about 5% near the sharp  $c_f$  peak at  $s \approx 3.7$  m (1.65 ft). Note that the factor of 10 increase in  $\Delta s$  causes no numerical problems either through transition [at  $s \approx 0.9$  m (0.3 ft)] or near the point of sudden application of adverse pressure gradient. Using the larger  $\Delta s$  reduces computing time for the Lewis-Gran-Kubota case by a factor of 4. The reason the reduction in computing time is less than a factor of 10 is because the number of iterations increases from one or two to an average of five or six.

### Discussion

Based on these results, we have finally solved what has been a long-standing problem of limited stepsize for boundary-layer computations which use advanced turbulence-model equations. Generalization of the method is straightforward for a block tridiagonal system such as that used in many three-dimensional boundary-layer programs. The reduction in computing time attending the increase in maximum permissible  $\Delta s$  is as dramatic in the three-dimensional case as it is in the two-dimensional case.<sup>6</sup>

### Acknowledgment

This research was sponsored by the NASA Ames Research Center under Contract NAS2-8192.

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